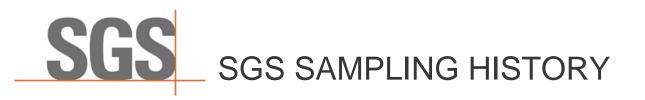
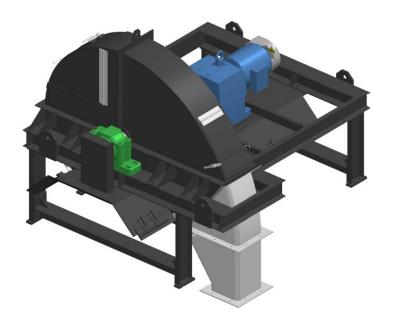
SGS MECHANICAL SAMPLING SYSTEMS



WHEN YOU NEED TO BE SURE



SGS has manufactured and supplied mechanical sampling equipment (MSS) primarily in Australia, Asia, and Europe for over 20 years. SGS cross-belt samplers are re-engineered and the design significantly updated and improved supply superior, yet cost effective systems.

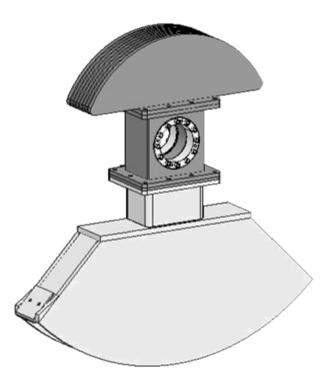




- Available to fit any size, speed or capacity belt, metric or imperial
- Large range of top sizes can be accommodated
- Heavy duty cutter and enclosure construction
- Detailed engineering of mechanical components
- Direct drive
- Fluid shear brake
- Adjustable skirt-boards
- Non-stick spiral enclosure
- Closed loop speed and position control available
- Hydraulic drive options available

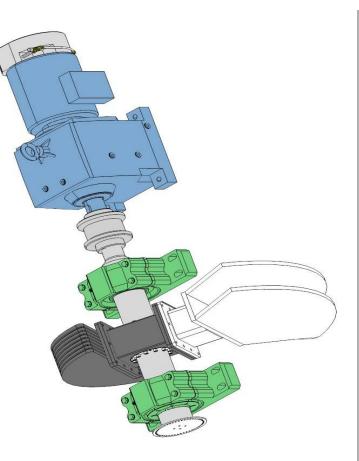


- Ultra heavy duty components for long life
- Engineered to minimize weight without compromising sample integrity
- Engineered to accommodate worst case loading
- Conservative safety factors
- Standard stainless steel cutter body
- Counterweighted for balance and increased inertia
- Counterweight eliminates the forces on the customer's structure resulting from an eccentric cutter load if no counter weight is used
- Belt wiper ensures complete increment collection



SGS CAREFULLY SIZED COMPONENTS

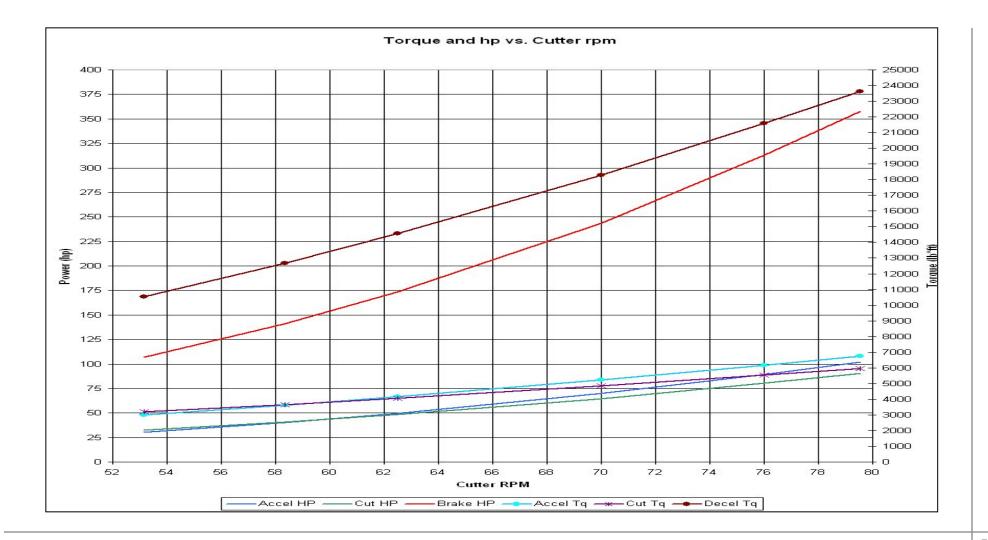
- Drive components custom selected for each application
- All components selected for durability
- Components are oversized to handle the harshest conditions
- Shock loading and fatigue are primary design considerations



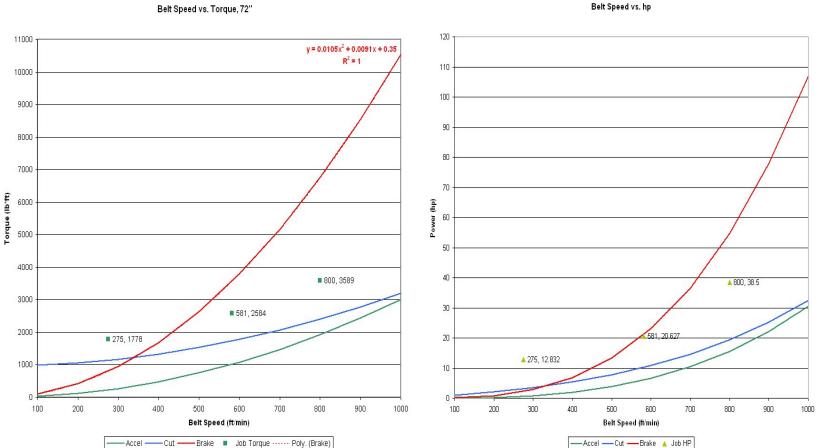


A	В		DE	F	G	H	J	к	L	M	N	0
1 Generic 48in Primary		8/17/2009										
2												
3 English/Imperial Units												
4		=Calculated cells										
5	Number	=Input cells										
6								1				
7 Belt and Material Data	Inputs		Geometric Results			Motion, Torque, and Power		1				
8		2							-	-		
9 Belt Width (b)	48	in	Cutter Radius (r_)	27.751	in	Total Cucle Time (estimated)	0.591	-		-		
10 Belt Capacity	2574		Cutter Angle (a.)		deg	Minimum Tip Speed of Cutter (min 1.5x of belt speed)	1200.000			-		
11 Belt Speed		ft/m	Clearance Badius (r.,)	10.789		Actual Max Tip Speed of Cutter	1243.771		green= ok, red			
12 Angle Of Idler (β)			Cutter Inside Vidth (v.;)		in	Full Speed BPM of cutter (gearbox output)	85.597		T	= check gear i	auo	
		degrees							-			
13 Roller Width plus air gap between rollers (I)	17.5	1225	Cutter Outside Vidth (w		in	Full Speed Angular Velocity	8.964	radians/s	∞=2π*rev/min	1		
14 Conveyor Incline Angle	0	degrees	Overall Shaft Length	27.418								
15 (if positive material is ascending)			Shaft Dia	4	in							
16						Acceleration Phase ($\omega_1 \cdot \omega_{**}$)						
17 Bulk Material Characteristics						Acceleration angle in radians	1.169	radians	0			
18 Surcharge Angle (α)	25	degrees	Total Rotational Mass			Angular Acceleration	34.355	radians/s^2	α = (co ^2 - co	o.^2)/(20)	0.0	- C.
19 Nominal Top Size	2	in	Cutter Rotating Mass	902.19	Ib	Time to accelerate to full speed	0.261	s	t = (co coc	s)/a.	1.1.2	
20 ASTM Factor	1.5	:1	Motor Rotor Mass	0.25	Ib	Acceleration Torque at main shaft	1228.676	Ib"ft	T-Ia (convert	t lbm to lbf!!!)	14	4744.1171
21			Coupling Rotating Mass	1.00		Acceleration HP	20.032		Hp = (T BPN			
22 Construction Materials			Shaft Mass	99.58						-		
23 Cutter			Total=	903.44						-		
24 Cutter Side Material Thickness	0.75	in	Total=	303.44	10	Constant Speed Phase (00)		-		-		
								-	-	L		
25 Steel Density (304 SS)	0.289	Ibłin^3				Ratio of Cutter tip speed to Belt speed	1.555	19 (Add)	Must be great	er than 1.5		
26			Inertia			Weight of sample	53.625					
27 Other Parts (hub, counterweight, spacer)			Cutter		Ib*ft*2	Angle over which sample is cut (cut angle=cutter angle)		radians		2		
28 Thickness	0.5		Motor (rotor and geartrain)		Ib*ft^2	Time to sweep coal off belt	0.174		2	-		
29 Density (304 SS)	0.289	Ib/in^3	Brake		lb*ft^2	Sample CG loc in y-axis (approx)	-21.646		/			
30	1		Coupling	0.050	Ib*ft^2	Velocity of CG of sample @∞	970.141	ft/min=	16,169	ft/s	v=d/t	
31 Motion Characteristics			Shaft	1.383	Ib*ft*2	Acceleration of sample	185.879	ft/s^2	a = v/t	Using t/2 bea	cause the	entire sa
32 Angle to accelerate cutter to full speed	67.000	degrees	Total Botational Inertia	1151.612	Ib"ft"2	Mass of coal buildup on cutter side	248.782	Ibm		2		1
33 Stopping angle		degrees	Cutter Rotating Mass	902.19		Deceleration of built up coal	153.280			Using t/2 bee	cause the	entire by
34 Overlap between sample cut and decel		degrees				Friction of built up coal on belt	124.391					
35 Angle over which sample is cut		degrees				Force of Coal buildup on cutter side	1308.656					
36 Steady State Angle (pre cut)		degrees	Bearing Reaction Forces			Torque to accelerate sample	558.394	1.001	F = ma = (Wt/3	22.21 * *		
37 Gear Reduction ratio	20,620		B _k =	0.00		Friction of coal to walls of outter	53.625		1 ma - (was	72.2) a		
38 Drive RPM from Motor	1765.000		Bu=	841.40		Friction of sample sliding on the belt	26.813					
	1765.000	DEM		041.40	IDF				-			
39						Friction from the cutter wiper	30.000		-			
40 Component Friction Coefficients			B=	-1433.05		Torque due to friction	255.400					
41 Friction Coefficient coal against the cutter 4	1.000		B.,=	-1743.59	Ibf	hp to accelerate sample	13.268					
42 Friction Coefficient coal sliding on the belt 4	0.500					Torque to accelerate and cut the sample	813.794	lb"ft	9765.53	lb*in		l.
43 Wiper Friction	5.000	Ibs/in of wiper										
44	1											
45						Braking Phase (ω_{**} - ω_{*})						
46 Shaft Physical Parameters						Stopping arc in radians	0.698	radians	8			12
47 Bearing Width	1.000	in				Angular deceleration	57.544	radians/s*2	α = (ω ^2 - α	a ^2)/(29)	5 S	
48 Driven Stub Shaft	3.000	in				Time to decelerate outter		sec	t = (co - coo) /			
49 Driving Stub Shaft	10.000					Torque to decelerate rotational components	2058.033		T-Ia (convert			696.396
49 Driving Stub Shart 50 Material Density		in Ibrin^3				hp to decelerate	2058.033		Leta (convert	. Iorn to ioriii)		99.808
50 Material Density 51 ou	73200					inp to decelerate	33.004		-			33.008
								-		Constant Inc.		
52 oy	31200					Maz Full CycleTorque	2058.033		24696.396			
53 E (Modulus of Elasticity)	28,500,000					Max Full Cycle Torque at Motor shaft	99.808	lb"ft	1197.691	in*lb		
54 Torsional Modulus G	11,500,000											
55 Permissible axial deflection	0.0015	radians			11 11	Ke per stop	1436.778	Ib"ft			1	
					1.0	Maz Power	33.554	-				
56 Maximum torsional deflection	0.08	degrees/ft of shaft				Max Power	33.994	пр				









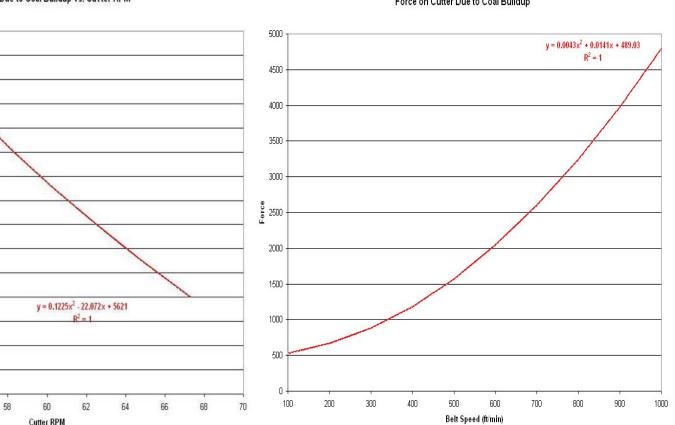
Belt Speed vs. Torque, 72"

⁸



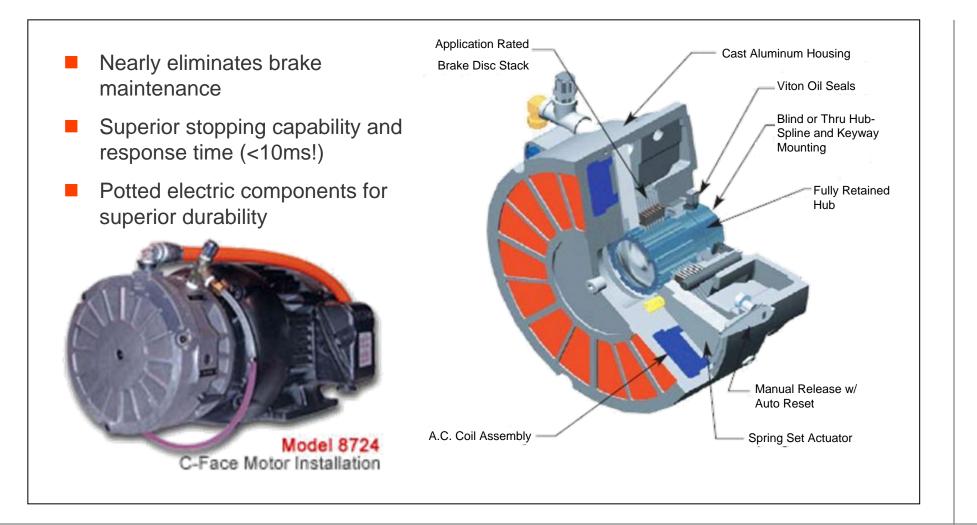
Force on Cutter Due to Coal Buildup vs. Cutter RPM

91 4730

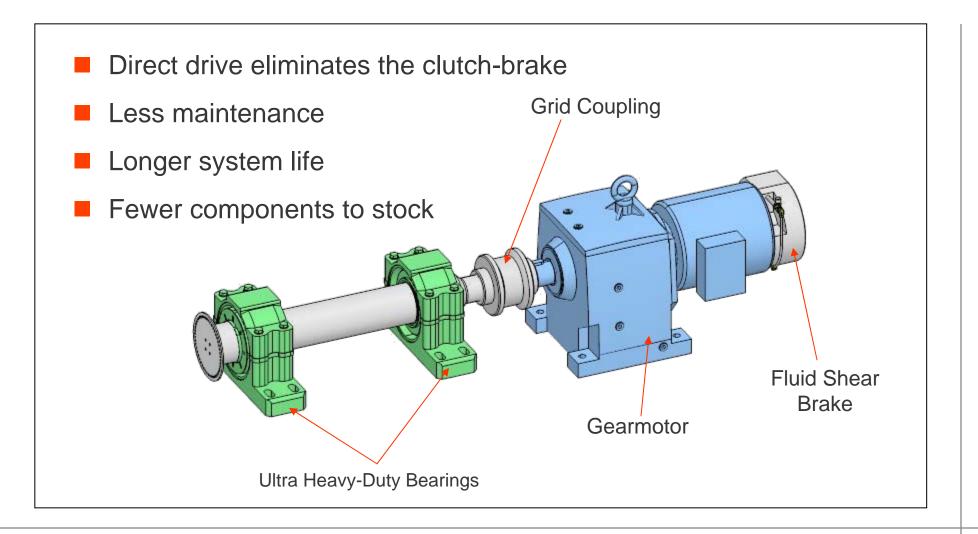


Force on Cutter Due to Coal Buildup

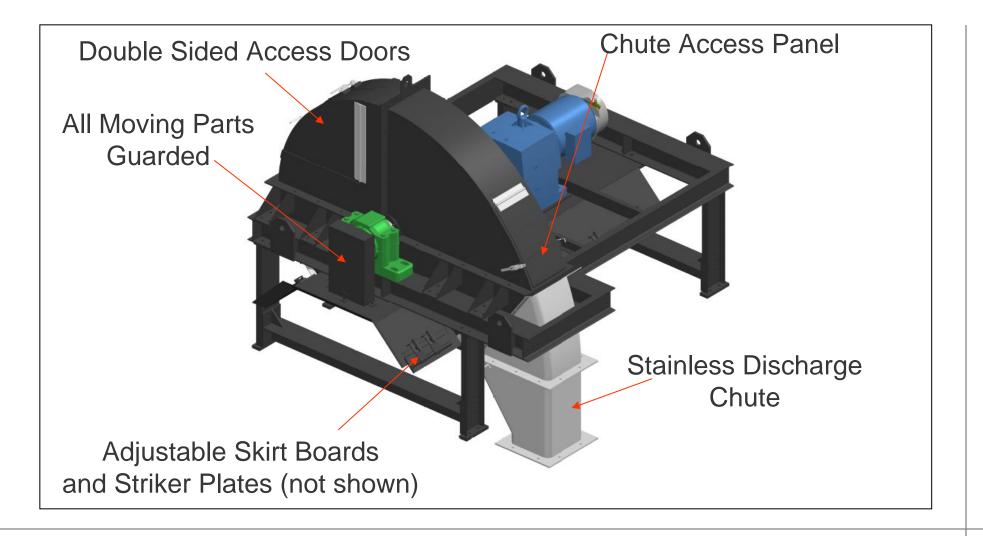








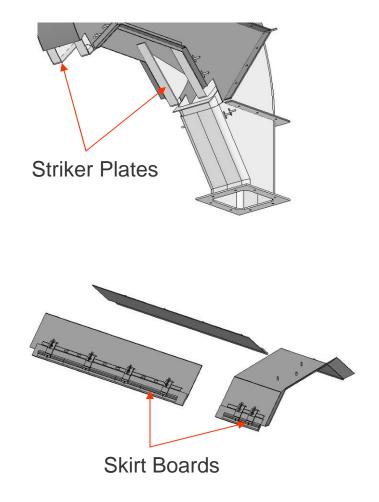




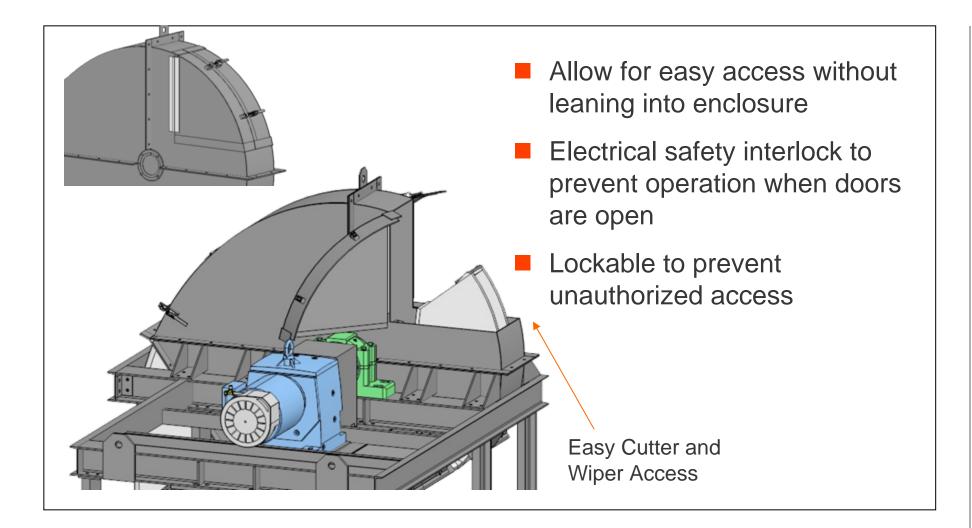


ADJUSTABLE SKIRT BOARDS AND STRIKER PLATES

- Collect all the increment, all the time
- Striker plates ensure only the material within the cutter is collected with the increment.
- Reduce bias, error, and dust
- No residual sample material left on belt
- Keep all non-sample material on the belt, eliminate spillage
- Conforms to ISO, ASTM and other international standards

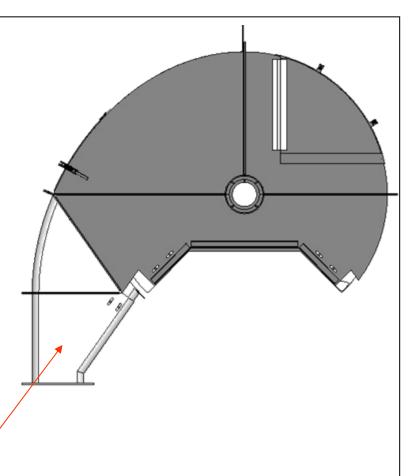


SGS SIDE HINGED ACCESS DOORS

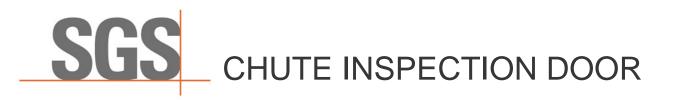




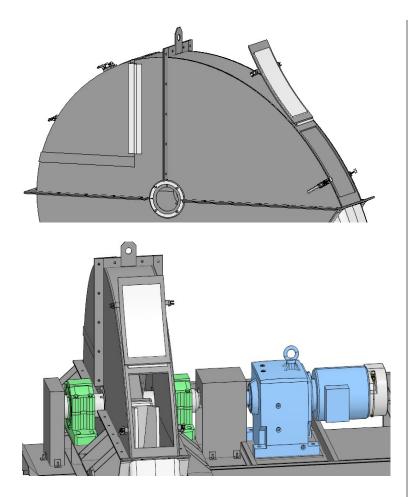
- Material impacts walls at 65° angle
- Prevents 'mushrooming' and sticking of sample on enclosure walls
- Maintains material velocity all the way to the discharge flange



Fully Stainless Steel Chute With 2" Radius Corners

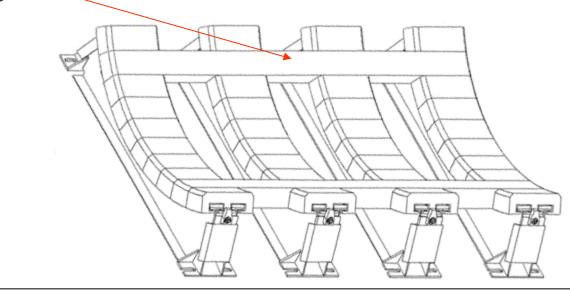


- Easy chute or cutter access
- Lockable latches
- Inner liner to keep inside of enclosure as flat as possible, reducing places for material to stick



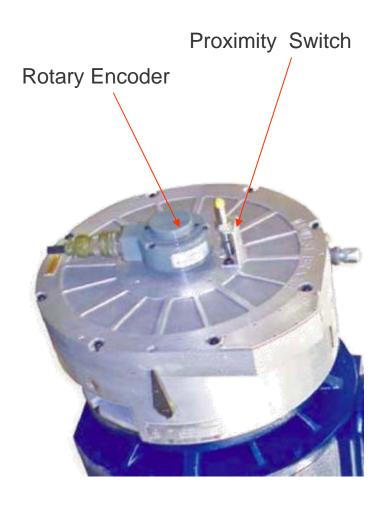


- Conforms belt perfectly to cutter trajectory (training idlers not shown)
 - Ensures that no material is left behind by the cutter
- Extra protection along skirt boards to prevent material loss as spillage





- Shaft encoder integrated with fluid shear brake or hydraulic motor
 - Allows for real-time feedback to control software
 - Operational speed can be verified and an alarm can be triggered if the correct speed is not reached in the designated angle
 - Any slowing through material can be monitored and accounted for, an alarm can be triggered if conditions change out of nominal parameters
 - Dynamic acceleration and braking control reduces stress on components
 - Park position and acceleration/braking curves precisely defined and monitored, no 'timer based' motion control
 - No 'tuning' necessary once system is set up





- Extreme duty drive
- Fully dynamic motion control
- Extreme torque and shock capacity
- 100% thermal control for full response in any environment
- Very low maintenance
- No need for a gearbox or separate brakes (holding brake and stopping brake included)
- Complete integrated solution
 - Motor
 - Power unit
 - Hydraulic controls
 - Hydraulic system health monitoring





- Fully stainless steel systems
- Wash-down rated equipment
- Explosion proof electrics (Hazardous environment conditions)
- Epoxy coated components (Highly corrosive applications)
- Galvanized framework
- Stainless hardware

SGS CROSS BELT SAMPLERS – DESIGN RULES

- Rule 1: The cutter aperture must travel at a 90 degree angle to the centerline of the belt being sampled.
- Rule 2: The cutter aperture width must be no less than three times the nominal top size of the material (3d).
- Rule 3: The arc formed from the leading edge to the trailing edge of the sidewalls must be sufficient for the cutter to cover the width of the material on the belt at full CEMA loading.
- Rule 4: The cutter must pass through the entire stream of material during one continuous operation with a minimum cutter velocity at the tip of the cutter near the belting of 1.5 times the velocity of the belt.
- Rule 5: Striker plates for prohibiting the entry of non-sample material must be installed on both the upstream and downstream sides of the cutter exit opening with gaps between the exiting cutter and striker plate held to no more than 10 mm.
- Rule 6: The underlying belting must be securely supported in such a way as to conform the belting to the cutter path (a circle).



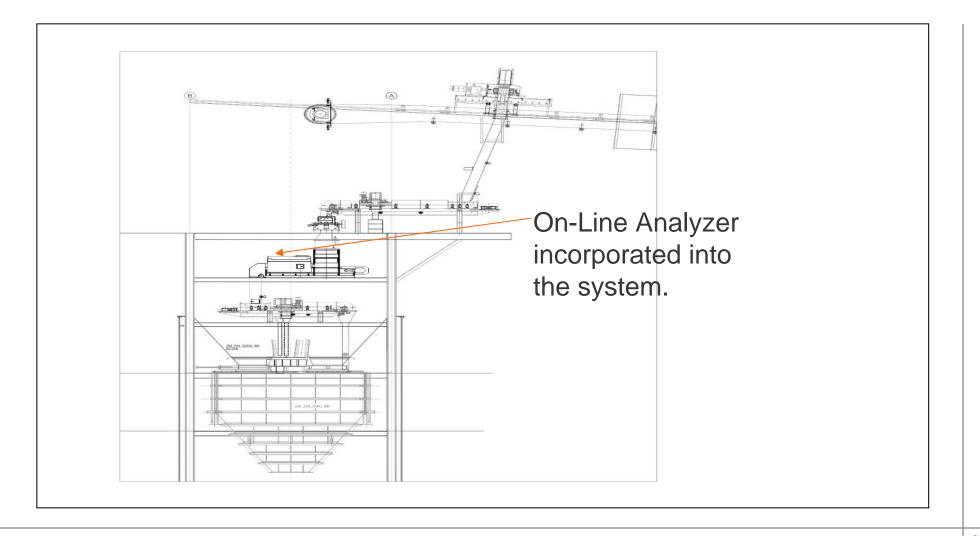
- Rule 7: The gap between the cutter side plates and the conveyor belting must not be greater than 10 mm at any point across the belt.
- Rule 8: The cutter must be equipped at the rear with an effective and durable wiper that cleanly scrapes the belting.
- Rule 9: The cutter must not be fitted with internal supports that could interfere with material entering or exiting the cutter.
- Rule 10: All cutters for belt sizes 900 mm and larger must be counterweighted. This avoids potential structural problems with the conveyor belt.
- Rule 11: All material delimited by the cutter and none other, must be included in the sample.
- Rule 12: No material shall remain in the cutter after the cutter has collected and discharged an increment. This is to be verified visually and by tests of the sampling ratio.



Fully turn-key sampling systems

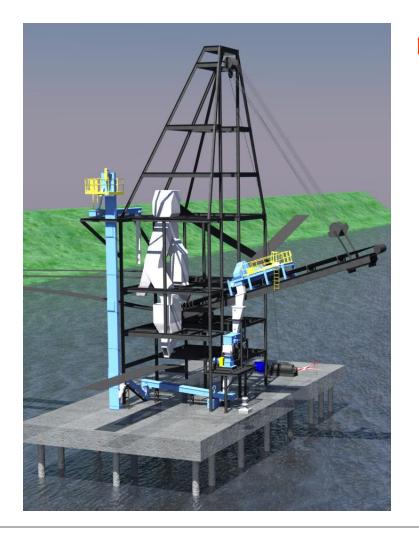
- Samplers (Primary, Secondary, Tertiary)
 - Cross Belt
 - Falling Stream
 - Auger
- Feed and Discharge Belts
- All Chute work
- Power, Instrumentation and Control Integration with Existing Systems







SGS COMPLICATED INTEGRATION OF A MSS INTO AN EXISTING FACILITIES



Primary sampler mounted on a loading boom that is raised and lowered.



- Bias Testing of Sample Systems
- Sampling System Inspection Services
- Engineering Studies to design and justify
- Field installation and Project management
- Integration of On-Line Analyzers into the system
- Operation and maintenance services

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WHEN YOU NEED TO BE SURE